

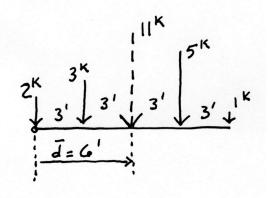
Parallel Force Resultant

The resultant is the single force that has the same effect as the group of forces.

$$\sum M = \sum (\mathbf{F} \times d) = \mathbf{R} \times \overline{d}$$

$$\sum \mathbf{F} = \mathbf{R}$$

$$\overline{d} = \frac{\sum (\mathbf{F} \times d)}{\sum \mathbf{F}}$$

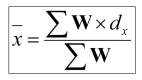


Centers

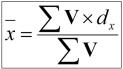
The point about which a body may be balanced.

This is the point of application of the resultant weight.

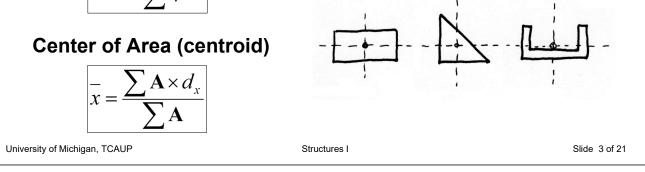
Center of Gravity



Center of Volume



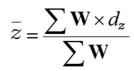
Tyrrell Photographic Collection, Powerhouse Museum

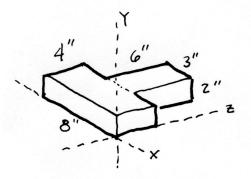


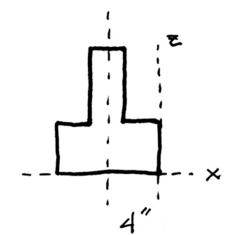
Center of Gravity (or Volume)

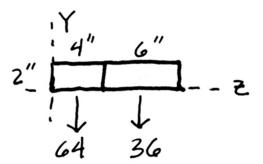
The Center of Gravity is located at the point defined by:

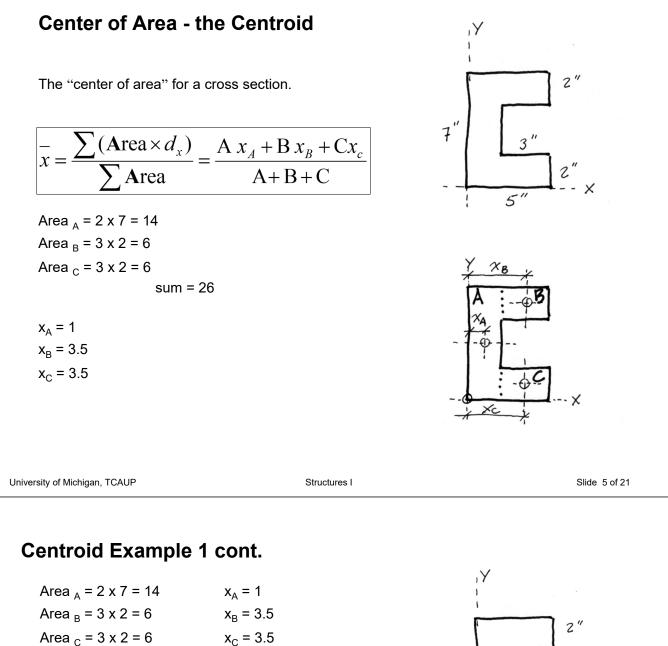








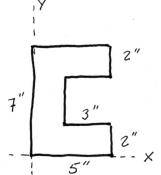


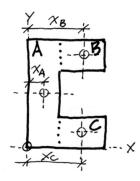


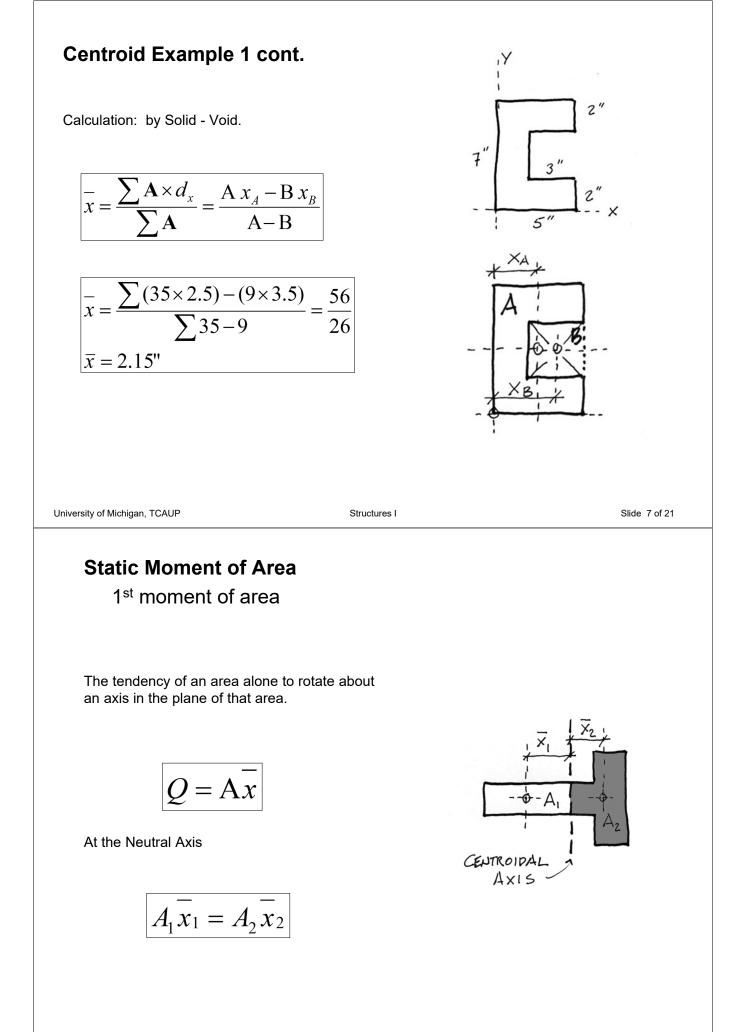
sum = 26

Calculation.

$$\overline{x} = \frac{\sum \operatorname{Area} \times d_x}{\sum \operatorname{Area}} = \frac{\operatorname{A} x_A + \operatorname{B} x_B + \operatorname{C} x_c}{\operatorname{A} + \operatorname{B} + \operatorname{C}}$$
$$\overline{x} = \frac{(14 \times 1) + (6 \times 3.5) + (6 \times 3.5)}{14 + 6 + 6}$$
$$\overline{x} = \frac{56}{26} = 2.15"$$

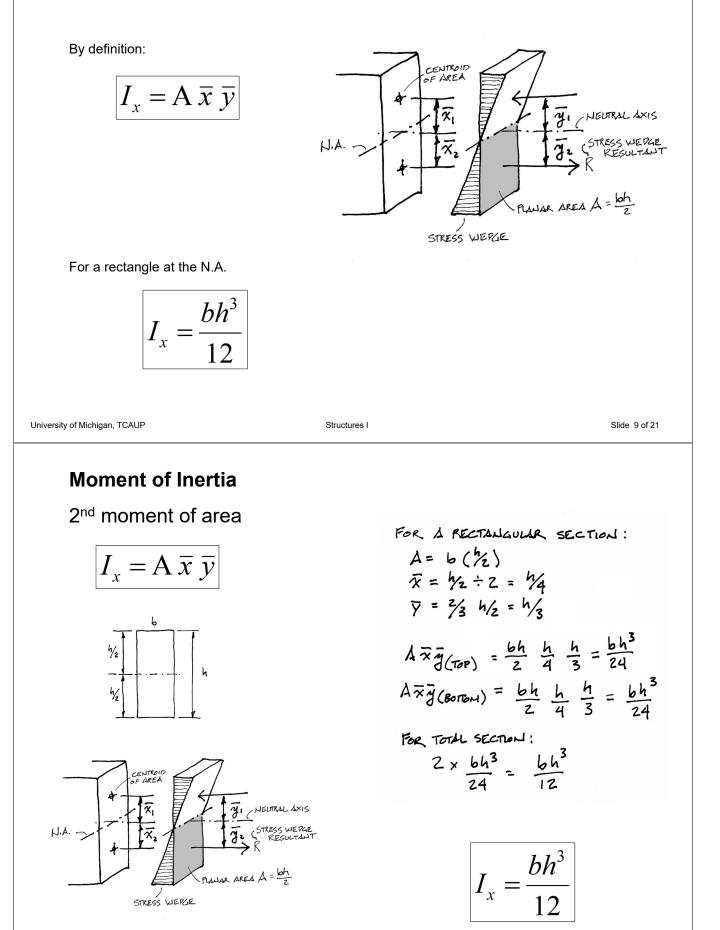


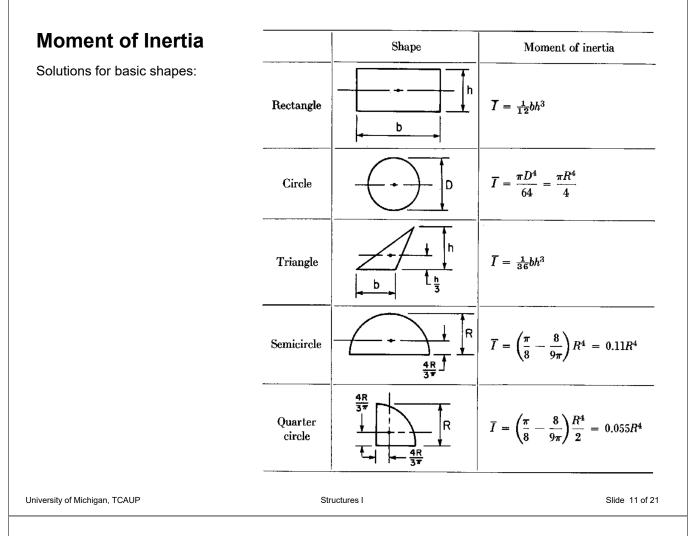




Moment of Inertia

2nd moment of area





Be

S 10×

S 12× 31.8 C 8×11.5 43.3 12.73 × 31.8 C 10×15.3 47.1 13.84 × 40.8 C 10×15.3 56.1 16.49

Moment of Inertia

Solutions for basic shapes:

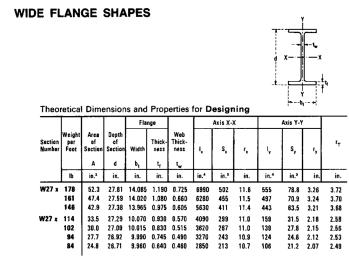
- Single Shapes •
- **Combination Shapes**

		С	S sł	nape	TION s and ies o	d cha	anne	ls	y; X 1	, y _p		x
Beam	Channel	Total	T + 1	AXIS X-X					AXIS Y-Y			
			Wt. Area	1	$S_1 = I/y_1$	$S_2 = I/y_2$	r	y 1	I	s	r	r ,
		Lb.	In. ²	In.⁴	In. ³	In. ³	In.	In.	ln.⁴	In. ³	In.	In.
10× 25.4 × 25.4	C 8×11.5 C 10×15.3	36.9 40.7	10.84 11.95	176 186	27.2 27.6	46.6 52.9	4.02 3.94	6.45 6.73	39.4 74.2	9.85 14.8	1.91 2.49	2.44 3.16

63.2 4.84 71.4 4.78 80.0 4.78

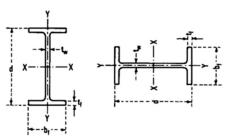
7.50 7.82 7.53 42.0 76.8 81.0 10.5 15.4 16.2 1.82 2.36 2.22 2.38 3.06 2.94

299 316 377 39.8 40.4 50.1



Section Properties

WIDE FLANGE SHAPES



				Flange			Axis X-X		Axis Y-Y					
Section Number	per of Foot Sectio	of	of of Section Section	ion Width		Web Thick- ness	l _x	S _x	r _x	l _y	S _y	ry	۲	
			d	₽ _f	ťf	^t w								
	lb	in.²	in.	in.	in.	in.	in.4	in.ª	in.	in.⁴	in.³	in.	in.	
W27 x	178	52.3	27.81	14.085	1.190	0.725	6990	502	11.6	555	78.8	3.26	3.72	
	161	47.4	27.59	14.020	1.080	0.660	6280	455	11.5	497	70.9	3.24	3.70	
	146	42.9	27.38	13.965	0.975	0.605	5630	411	11.4	443	63.5	3.21	3.68	
W27 x	114	33.5	27.29	10.070	0.930	0.570	4090	299	11.0	159	31.5	2.18	2.58	
	102	30.0	27.09	10.015	0.830	0.515	3620	267	11.0	139	27.8	2.15	2.56	
	94	27.7	26.92	9.990	0.745	0.490	3270	243	10.9	124	24.8	2.12	2.53	
	84	24.8	26.71	9.960	0.640	0.460	2850	213	10.7	106	21.2	2.07	2.49	

Theoretical Dimensions and Properties for Designing

Section Properties

PROPERTIES OF SAWN LUMBER SECTIONS

.__.×

Rectangular :

A = bd

 $I = db^{3}/12$

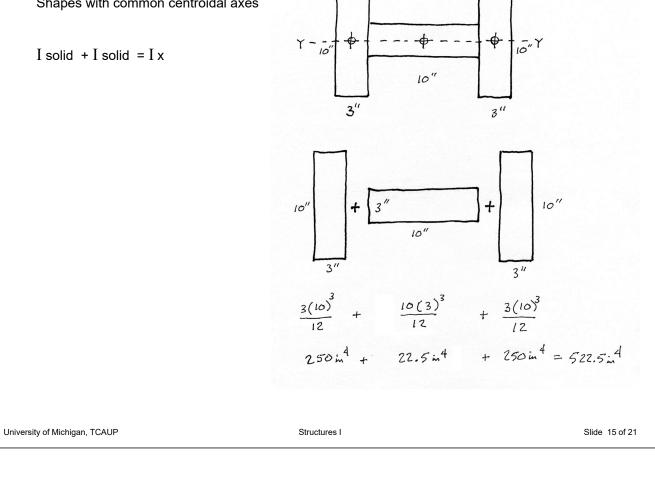
S = I/c

c = d/2 (maximum)

Actual Size	Area	I_x	S _x
$b \times d$	in. ²	in.4	in. ³
$3/4 \times 3\frac{1}{2}$	2.63	2.68	1.53
$" \times 5\frac{1}{2}$	4.13	10.40	3.78
$'' \times 7\frac{1}{4}$	5.44	23.82	6.57
$" \times 9\frac{1}{4}$	6.94	49.47	10.70
" $\times 11\frac{1}{4}$	8.44	88.99	15.83
$1\frac{1}{2} \times 3\frac{1}{2}$	5.25	5.36	3.06
$'' \times 5\frac{1}{2}$	8.25	20.80	7.56
$'' \times 7\frac{1}{4}$	10.88	47.64	13.14
$" \times 9\frac{1}{4}$	13.88	98.93	21.39
" $\times 11\frac{1}{4}$	16.88	177.98	31.64
$2\frac{1}{2} \times 3\frac{1}{2}$	8.75	8.93	5.10
$" \times 5\frac{1}{2}$	13.75	34.66	12.60
$'' \times 7\frac{1}{4}$	18.13	79.39	21.90
$" \times 9\frac{1}{4}$	23.13	164.89	35.65
$'' \times 11\frac{1}{4}$	28.13	296.63	52.73
$3\frac{1}{2} \times 3\frac{1}{2}$	12.25	12.50	7.15
$" \times 5\frac{1}{2}$	19.25	48.53	17.65
$'' \times 7\frac{1}{4}$	25.38	111.15	30.66
$" \times 9\frac{1}{4}$	32.38	230.84	49.91
$'' \times 11\frac{1}{4}$	39.38	415.28	73.83
	$\begin{array}{c} b \times d \\ \hline 3/4 \times 3\frac{1}{2} \\ " \times 5\frac{1}{2} \\ " \times 74 \\ " \times 94 \\ " \times 114 \\ \hline 1\frac{1}{2} \times 3\frac{1}{2} \\ " \times 5\frac{1}{2} \\ " \times 74 \\ " \times 94 \\ " \times 74 \\ " \times 94 \\ " \times 114 \\ \hline 2\frac{1}{2} \times 3\frac{1}{2} \\ " \times 5\frac{1}{2} \\ " \times 74 \\ " \times 94 \\ " \times 114 \\ \hline 3\frac{1}{2} \times 3\frac{1}{2} \\ " \times 74 \\ " \times 94 \\ \end{array}$	$\begin{array}{c ccccc} b \times d & \text{in.}^2 \\ \hline & 3/4 \times 3\frac{1}{2} & 2.63 \\ \hline & & \times 5\frac{1}{2} & 4.13 \\ \hline & & \times 7\frac{1}{4} & 5.44 \\ \hline & & \times 9\frac{1}{4} & 6.94 \\ \hline & & \times 11\frac{1}{4} & 8.44 \\ \hline & 1\frac{1}{2} \times 3\frac{1}{2} & 5.25 \\ \hline & & \times 5\frac{1}{2} & 8.25 \\ \hline & & \times 7\frac{1}{4} & 10.88 \\ \hline & & \times 9\frac{1}{4} & 13.88 \\ \hline & & \times 9\frac{1}{4} & 13.81 \\ \hline & & \times 9\frac{1}{4} & 23.13 \\ \hline & & & \times 9\frac{1}{4} & 23.13 \\ \hline & & & \times 11\frac{1}{4} & 28.13 \\ \hline & & & 3\frac{1}{2} \times 3\frac{1}{2} & 12.25 \\ \hline & & & \times 5\frac{1}{2} & 19.25 \\ \hline & & & \times 7\frac{1}{4} & 25.38 \\ \hline & & & & \times 9\frac{1}{4} & 32.38 \\ \hline \end{array}$	b × din.2in.4 $3/4 \times 3\frac{1}{2}$ 2.632.68" × 5\frac{1}{2}4.1310.40" × 745.4423.82" × 946.9449.47" × 1148.4488.99 $1\frac{1}{2} \times 3\frac{1}{2}$ 5.255.36" × 5\frac{1}{2}8.2520.80" × 7410.8847.64" × 9413.8898.93" × 11416.88177.98 $2\frac{1}{2} \times 3\frac{1}{2}$ 8.758.93" × 11416.88177.98 $2\frac{1}{2} \times 3\frac{1}{2}$ 13.7534.66" × 7418.1379.39" × 9423.13164.89" × 11428.13296.63 $3\frac{1}{2} \times 3\frac{1}{2}$ 12.2512.50" × 5\frac{1}{2}19.2548.53" × 7425.38111.15" × 9432.38230.84

Moment of Inertia

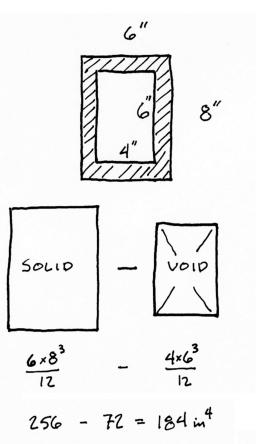
Shapes with common centroidal axes



Moment of Inertia

Shapes with common centroidal axes

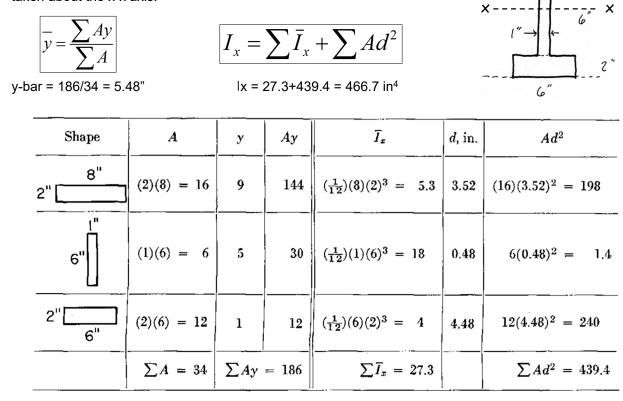
I solid - I void = I x



Moment of Inertia

The Transfer Equation or Parallel Axis Theorem,

taken about the x-x axis:



Structures I

ly = 121.8+0 = 121.8

y-bar = 186/34 = 5.48"

lx = 27.3+439.4 = 466.7 in⁴

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Moment of Inertia

University of Michigan, TCAUP

The Transfer Equation or Parallel Axis Theorem:

$$I_y = \sum \bar{I}_y + \sum Ad^2$$

Taken about the y-y axis:

8″

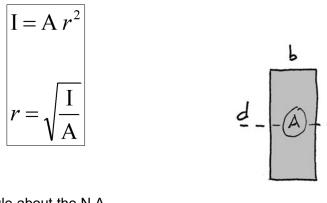
2″

Shape	A	\overline{I}_Y	d	Ad ²
2" 	16	$(\frac{1}{12})(2)(8)^3 = 85.3$	0	0
6"	6	$(\frac{1}{12})(6)(1)^3 = 0.5$	0	0
2"	12	$(\frac{1}{12})(2)(6)^3 = 36.0$	0	0
		$\sum \overline{I}_Y = 121.8$		0

SUMMARY: $I_x = 466.7 \text{ in}^4$ $I_y = 121.8 \text{ in}^4$

Radius of Gyration

The distance from the centroid where all area could be collected to yield an equivalent Moment of Inertia.



r = 0.289 d

for a rectangle about the N.A

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Structures I

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%A

Radius of Gyration

The larger the radius of gyration, the more resistant the section is to buckling.

The areas in the table below are constant, while diameters increase.

OD	ID	t	Α	r
3.57	0.00	1.78	10.00	0.89
3.71	1.00	1.35	10.00	0.96
4.09	2.00	1.05	10.00	1.14
4.66	3.00	0.83	10.00	1.39
5.36	4.00	0.68	10.00	1.67
6.14	5.00	0.57	10.00	1.98
6.98	6.00	0.49	10.00	2.30
7.86	7.00	0.43	10.00	2.63
8.76	8.00	0.38	10.00	2.97
9.68	9.00	0.34	10.00	3.30
10.62	10.00	0.31	10.00	3.65

$$Pcr = \frac{\pi^2 E}{\left(\frac{KL}{r}\right)^2}$$

